

Test Centre for Solar Thermal  
Components and Systems

1. Summary

|          |  |                     |            |
|----------|--|---------------------|------------|
| Company: | <b>Thermomax Ltd.</b><br>Balloo Crescent<br>Bangor, BT 19 7UP<br>United Kindom | Report no.:         | 110-06/D   |
|          |  | Report date:        | 03.11.2006 |
| Type:    | <b>HP 200 30</b>   | Serial no.:         | MB 08617   |
|          |  | Year of production: | 2006       |

The following results were obtained from a test of the thermal performance of a solar collector according to **EN 12975-2:2006**. They apply to the collector described more precisely in the test report no. 110-06/D and to the tests and procedures described herein.

**Description of the collector**

|                         |                             |                             |                             |
|-------------------------|-----------------------------|-----------------------------|-----------------------------|
| Type                    | Evacuated tubular collector | Aperture area               | 3.229 m <sup>2</sup>        |
| Length/Width/Height     | 2005 / 2127 / 97 mm         | Absorber area               | 3.021 m <sup>2</sup>        |
| Max. operation pressure | 8 bar                       | Gross area                  | 4.265 m <sup>2</sup>        |
| Weight, empty           | 75.1 kg                     | Recommended flow rate       | 60..150 kg/m <sup>2</sup> h |
| Heat transfer fluid     | Polypropylene               | Thickness of absorber sheet | 0.12 mm                     |
|                         |                             | Number of tubes             | 30                          |

**Test results**

**Coefficients of efficiency**

(determined outdoor)

$$\eta = \eta_0 - a_1 \cdot (t_m - t_a)/G - a_2 \cdot (t_m - t_a)^2/G$$

Based on: aperture area      absorber area

|            |  |  |
|------------|--|--|
| $\eta_0 =$ | 0.727                                  | 0.778                                  |
| $a_1 =$    | 0.85 W/m <sup>2</sup> K                | 0.91 W/m <sup>2</sup> K                |
| $a_2 =$    | 0.0093 W/m <sup>2</sup> K <sup>2</sup> | 0.0100 W/m <sup>2</sup> K <sup>2</sup> |

**Incident angle modifier**

(determined outdoor)

| proj. angle of incidence $\theta$     | 0°   | 10°  | 20°  | 30°  | 40°  | 50°  | 60°  |
|---------------------------------------|------|------|------|------|------|------|------|
| $K_{\theta b, trans}(\theta_{trans})$ | 1.00 | 1.00 | 1.02 | 1.03 | 1.01 | 0.94 | 0.80 |
| $K_{\theta b, long}(\theta_{long})$   | 1.00 | 1.00 | 0.99 | 0.98 | 0.96 | 0.92 | 0.86 |
| $K_{\theta d} =$                      | 0.88 |      |      |      |      |      |      |

**Power output per collector unit**

| $T_m - T_a$ | 400 W/m <sup>2</sup> | Irradiance<br>700 W/m <sup>2</sup> | 1000 W/m <sup>2</sup> |
|-------------|----------------------|------------------------------------|-----------------------|
| 10 K        | 909 W                | 1614 W                             | 2319 W                |
| 30 K        | 830 W                | 1535 W                             | 2240 W                |
| 50 K        | 727 W                | 1432 W                             | 2137 W                |

**Peak power per collector unit**

**2349 W<sub>peak</sub>**

at  $G = 1000 \text{ W/m}^2$  and  $t_m - t_a = 0 \text{ K}$

**Pressure drop** (water, 20 °C)

$\Delta p = 0.8 \text{ mbar}$   
 $\Delta p = 4.4 \text{ mbar}$

at  $\dot{m} = 50.3 \text{ kg/h}$

at  $\dot{m} = 130.0 \text{ kg/h}$

**Thermal capacity** (calculated)

$c = 4.2 \text{ kJ/(m}^2\text{K)}$

$C = 13.6 \text{ kJ/K}$

**Stagnation temperature**

$t_{stg} = 183.6 \text{ °C}$

at  $G_S = 1000 \text{ W/m}^2$  and  $t_{as} = 30 \text{ °C}$

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Dipl.-Ing. C. Lampe, deputy head of Test Centre-EN